

BAE 820 Physical Principles of Environmental Systems

Stokes' law and Reynold number

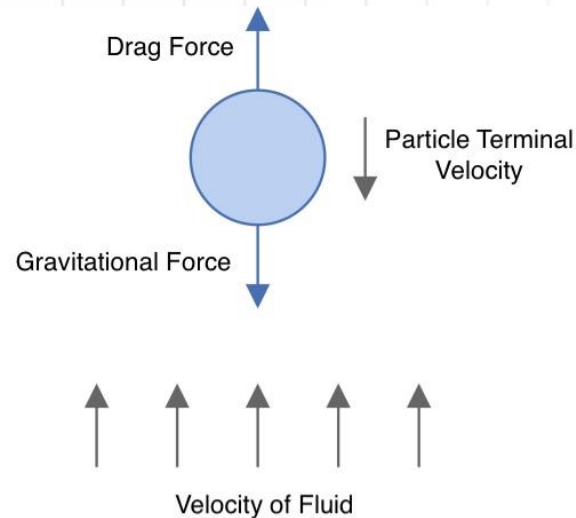
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The motion of a particle in a fluid environment, such as air or water

$$m \frac{dV}{dt} = F(t) - F_d - \frac{1}{2} \frac{4}{3} \pi r^3 \rho \frac{dV}{dt} - \frac{4}{3} \pi r^3 \frac{dp}{dr} - F_B$$

Where,

- $F(t)$ is external force.
- F_d is the resistance of the medium for a constant velocity.
- The third term represents the resistance to accelerated motion of the particle.
- The fourth term represents the resistance due to the pressure gradient across the particle.
- The fifth term is the Basset force (F_B), which represents the energy recovery expended in setting the medium itself in motion.



Newton's resistance law

$$F_d = \frac{1}{2} c_d \rho v^2 A$$

- F_d is the drag force of the fluid medium
- ρ is the mass density of the fluid
- v is the speed of the object relative to the fluid
- A is the projected area, for a sphere, $A = \pi r^2$
- c_d is the dimensionless drag coefficient, which is not a constant but varies as a function of speed, flow direction, object position, object size, fluid density and fluid viscosity.

Stoke's law

Under laminar flow conditions (undisturbed flow), the force of viscosity on a small sphere moving through a viscous fluid is given by

$$F_s = 6\pi r\eta v = \xi v$$

- F_s is the frictional force – known as Stokes' drag.
- η is the dynamic viscosity, kg / (m·s).
- r is the radius of the spherical object, m.
- v is the speed of the object relative to the fluid, m/s.
- ξ is the friction factor, kg·m/s², in which 2/3 is due to skin friction, and 1/3 is due to pressure forces.

Under laminar flow conditions, setting the two expressions equal to each other we get

$$6\pi r\eta v = \frac{1}{2} c_d \rho v^2 A$$

Therefore,

$$C_d = 12 \eta / r \rho v = 24 \eta / D \rho v$$

- D is the diameter of the spherical object

Define Reynold number,

$$R_e = D \rho v / \eta = D v / \nu$$

- Where $\nu = \eta / \rho$, is called the kinematic viscosity, m^2/s .

So, the dimensionless drag coefficient

$$C_d = 24 / R_e$$

The Reynold number (R_e)

- R_e is a dimensionless quantity that is used to characterize different flow regimes, such as laminar or turbulent flow.
- R_e can be defined as the ratio of inertial forces to viscous forces and quantifies the relative importance of these two types of forces for given flow conditions.

$$R_e = D\rho v / \eta = \frac{\rho v \pi D^3}{\eta \pi D^2} = \frac{\textit{inertial forces}}{\textit{viscous forces}}$$

- R_e can be used to determine dynamic similitude between two different cases of fluid flow. It frequently arise when performing scaling of fluid dynamics problems.

The Reynold number (R_e)

- R_e can be defined for several different situations where a fluid is in relative motion to a surface.

	Laminar flow	Turbulent flow
Fluid around sphere object	$R_e < 1$	$R_e > 700$
Fluid in pipe	$R_e < 2,100$	$R_e > 10,000$

Magnitude of the kinematic viscosity and R_e

Fluid	η (Pa s) at 20°C	ν (m ² /s) at 20°C	R_e (when $D=1\text{m}$ and $v=1\text{m/s}$)
Air	1.8×10^{-5}	1.5×10^{-5}	$\sim 7 \times 10^4$
Water	1×10^{-3}	1×10^{-6}	$\sim 1 \times 10^6$

- Most surface water systems are turbulent.
- Groundwater flow is nearly always laminar since the characteristic length scale for groundwater systems is the pore scale.

R_e of spherical particles falling at their terminal velocities in air

D_p (μm)	R_e (at 298 K and 1 atm)
20	0.02
60	0.4
100	2
300	20

- Thus, for particles smaller than about 20 μm in diameter, Stokes' law is an accurate formula for the drag force exerted by the air.

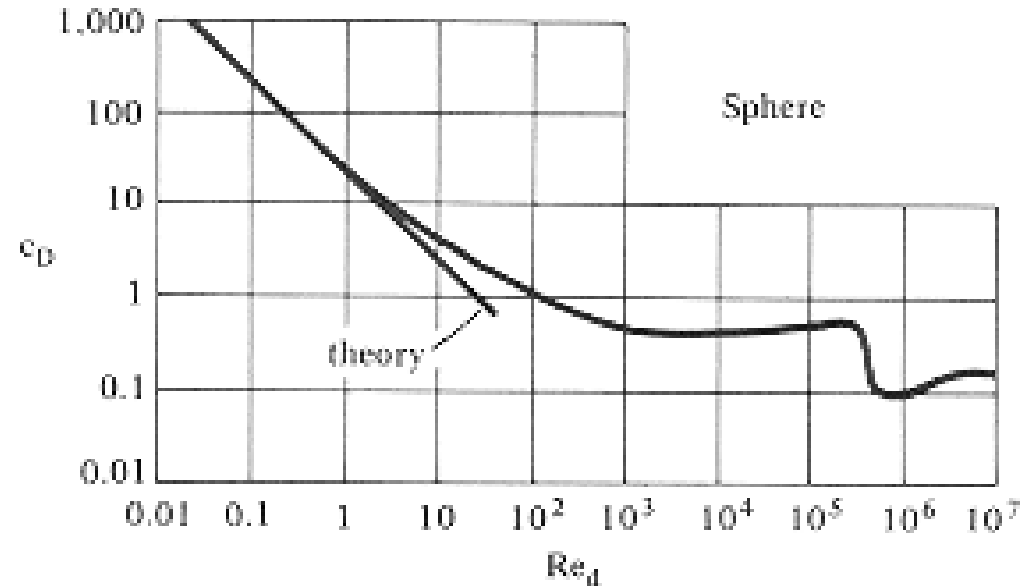
Relationship between R_e and c_d

- For $R_e < 1$, $c_d = 24/R_e$
- For $1 < R_e < 1,000$, transitional flow,
 $c_d = 24/R_e + 3/\sqrt{R_e} + 0.34$

An approximate form is

$$c_d = b/R_e^a$$

- For $1,000 < R_e < 100,000$,
 $c_d = \text{constant} \sim 0.44$
- At $R_e \sim 500,000$, c_d drops by a factor of about five and a new flow situation has been established.



From: Wegener, "What Makes Airplanes Fly?"
Springer-Verlag (1991)

The motion of a particle in a fluid environment

$$m \frac{dV}{dt} = F(t) - F_d$$

What is the settling velocity (v_t)?

- When Stokes law is obeyed,

$$6\pi r \eta v_t = F(t) = \frac{4}{3} \pi r^3 (\rho_p - \rho_l) g \Rightarrow v_t = \frac{2r^2 (\rho_p - \rho_l) g}{9\eta}$$

- When Stokes law is not obeyed,

$$\frac{1}{2} c_d \rho_l v_t^2 A = F(t) = \frac{4}{3} \pi r^3 (\rho_p - \rho_l) g \Rightarrow v_t^2 = \frac{8r (\rho_p / \rho_l - 1) g}{3 c_d}$$

A trail and error procedure

1. Calculate v_t using turbulent or laminar flow equation based on assumptions from your best guess.
2. Plug v_t into $Re = D\rho v_t / \eta$, check Re . If your initial assumption is valid, then problem is solved.
3. Make new assumptions based on calculated Re . Use Re to calculate c_d .
4. Use c_d to calculate v_t .
5. If v_t is not equal to previous v_t , using the new value for v_t and repeat from step 2.

Problem Solving #1

- Calculate settling velocity (v_t) of a particle with diameter=10 μm and 100 μm respectively, in air at 298 K and 1 atm. Assuming density of the particle $\rho_p=1000\text{kg/m}^3$

Problem Solution #1

$$\rho_p = 1000 \text{ kg/m}^3; \rho_l = 1.20 \text{ kg/m}^3$$

- For $D_p = 10 \mu\text{m}$

Assume laminar condition,

$$v_t = \frac{2r^2(\rho_p - \rho_l)g}{9\eta} = \frac{2(10/2 \times 10^{-6})^2(1000 - 1.20) \times 9.8}{9(1.8 \times 10^{-5})} = 3.02 \times 10^{-3} \text{ m/s}$$

Check

$$R_e = D_p \rho_l v_t / \eta = \frac{(10 \times 10^{-6})(1.20) \times 3.02 \times 10^{-3}}{(1.8 \times 10^{-5})} = 2.01 \times 10^{-3} \ll 1$$

So, the assumption of laminar condition is valid.

- For $D_p = 100\mu\text{m}$

Assume laminar condition,

$$v_t = \frac{2r^2(\rho_p - \rho_l)g}{9\eta} = \frac{2(100/2 \times 10^{-6})^2(1000 - 1.20) \times 9.8}{9(1.8 \times 10^{-5})} = 0.30 \text{ m/s}$$

Check

$$R_e = D_p \rho_l v_t / \eta = \frac{(100 \times 10^{-6})(1.20) \times 0.3}{1.8 \times 10^{-5}} = 2 > 1$$

So, the assumption of laminar condition is not valid.

In transitional flow,

$$c_d = 24/R_e + 3/\sqrt{R_e} + 0.34 = 24/2 + 3/\sqrt{2} + 0.34 = 14.5$$

$$v_t^2 = \frac{8r(\rho_p/\rho_l - 1)g}{3cd} = \frac{8 \times 100 / 2 \times 10^{-6} \times (1000/1.2 - 1) \times 9.8}{3 \times 14.5} = 0.075$$

$$v_t = 0.27 \text{ m/s}$$

Not same with the initial value of v_t , so continue the procedure,

$$R_e = D_p \rho_l v_t / \eta = \frac{(100 \times 10^{-6})(1.20) \times 0.27}{1.8 \times 10^{-5}} = 1.8 > 1$$

$$c_d = 24/R_e + 3/\sqrt{R_e} + 0.34 = 24/1.8 + 3/\sqrt{1.8} + 0.34 = 15.9$$

$$v_t^2 = \frac{8r(\rho_p/\rho_l - 1)g}{3cd} = \frac{8 \times 100 / 2 \times 10^{-6} \times (1000/1.2 - 1) \times 9.8}{3 \times 15.9} = 0.068$$

$$v_t = 0.26 \text{ m/s}$$

Not same with the previous value of v_t , so continue the procedure,

$$R_e = D_p \rho_l v_t / \eta = \frac{(100 \times 10^{-6})(1.20) \times 0.26}{1.8 \times 10^{-5}} = 1.7 > 1$$

$$c_d = 24/R_e + 3/\sqrt{R_e} + 0.34 = 24/1.7 + 3/\sqrt{1.7} + 0.34 = 16.8$$

$$v_t^2 = \frac{8r(\rho_p/\rho_l - 1)g}{3c_d} = \frac{8 \times 100 / 2 \times 10^{-6} \times (1000/1.2 - 1) \times 9.8}{3 \times 16.8} = 0.065$$

$$v_t = 0.25 \text{ m/s}$$

Not same with the previous value of v_t , so continue the procedure,

$$R_e = D_p \rho_l v_t / \eta = \frac{(100 \times 10^{-6})(1.20) \times 0.25}{1.8 \times 10^{-5}} = 1.7 > 1$$

$$c_d = 24/R_e + 3/\sqrt{R_e} + 0.34 = 24/1.7 + 3/\sqrt{1.7} + 0.34 = 16.8$$

$$v_t^2 = \frac{8r(\rho_p/\rho_l - 1)g}{3c_d} = \frac{8 \times 100 / 2 \times 10^{-6} \times (1000/1.2 - 1) \times 9.8}{3 \times 16.8} = 0.065$$

$$v_t = 0.25 \text{ m/s}$$

Same with the previous value of v_t . At this point, we may consider the problem solved. $v_t = 0.25 \text{ m/s}$ can be the final answer.